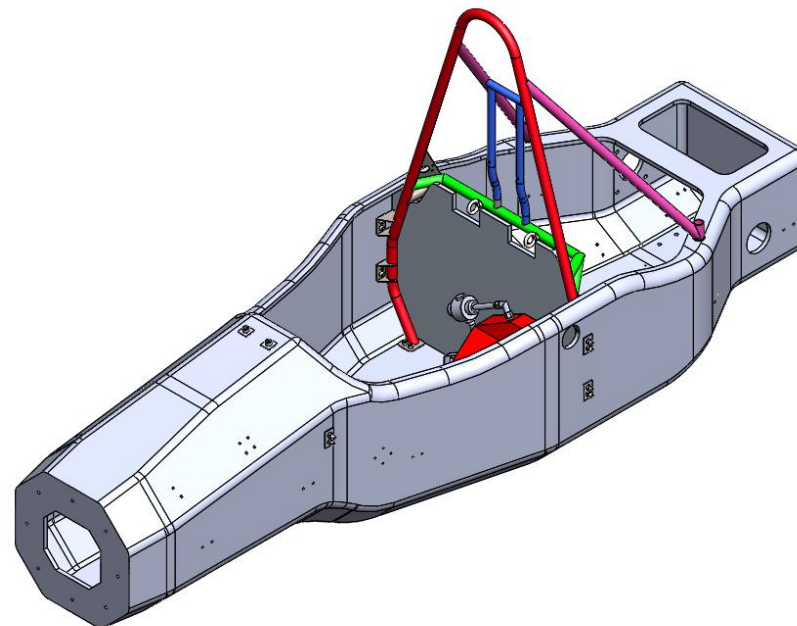


# SES Guidance for FSAEJ

## Monocoque



2025 v1.1 Compliant

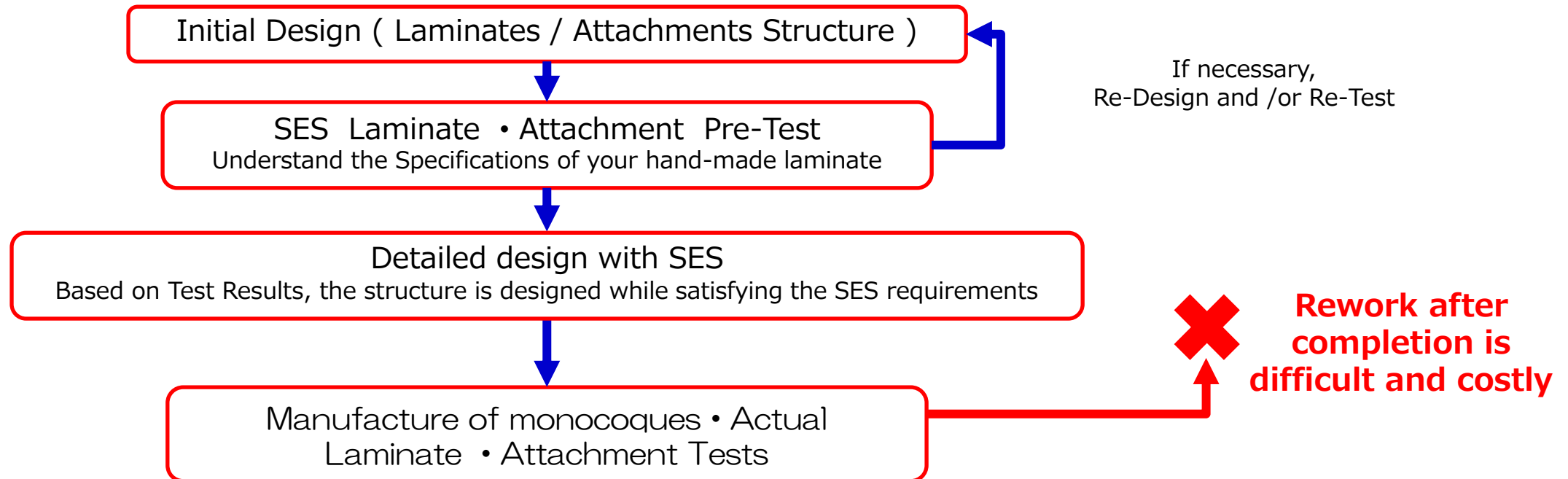
# How to describe Monocoque SES

- Design flow with SES / SESを用いた設計フロー

SES requires proof tests (Laminate Test and Attachment Test) for equivalence proof

⇒ By making good use of SES,

you can avoid the risk of fatal rework and non-compliance with rules.



If you test after manufacturing Monocoque and find that it lacks strength or does not comply with the rules, Recovery is more difficult than Steel Tube Frame, so use SES well!

- Basic Procedure of SES input / SES入力手順

① F.3.1-5 Tube Chassis -> Basic Info & Select [Tube] or [Composite]  
Define your Composite Portion in the Structure.



② F.4.3 Composite  
If necessary, duplicate [F.4.3 Composite] Sheet for Different or Additional Layout  
It's strongly recommended to **be completed** before proceeding to the next step.



③ Test section in F.7.9-10 Attachments & in F.8 Front Protection ( & in F.10-11 EV Accumulator )  
Sometime test results affect your Chassis design



④ F.7 Composite Chassis, Remaining F.7.9-10 Attachments (and F.10-11 EV Accumulator )



⑤ Remaining F.8 Front Protection  
Front Bulkhead section requires to complete FBHS section in F.7 Composite Chassis

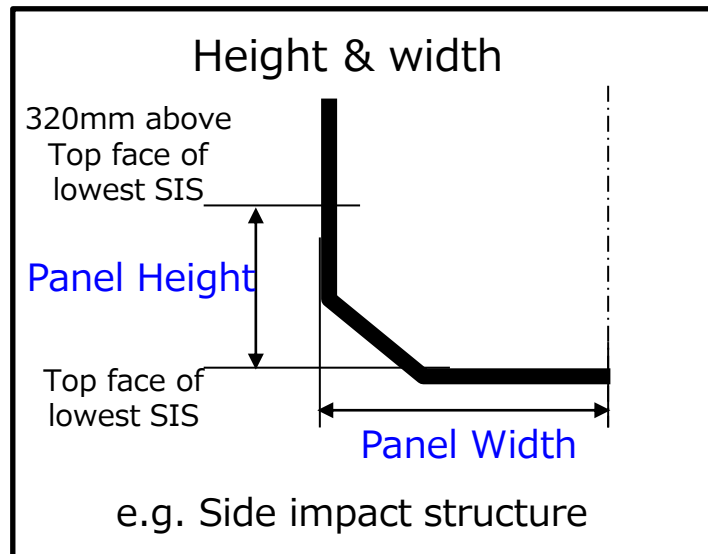


⑥ Fill in remaining BLANKs

\* Of course, BLANKs may be filled when possible.

- Equivalent Flat Panel Calculation (F.4.4) / フラットパネル換算

The EI of the monocoque is calculated as that of a flat panel with the same composition as the monocoque about the neutral axis of the laminate. The curvature of the panel and geometric cross section of the monocoque must be ignored for these calculations.



Note : Comply with F.4.4 for the following calculations

- Front Bulkhead Support Structure  
Vertical wall must have EI more than ONE Baseline steel tube.
- Side Impact Structure  
Vertical wall must have EI more than TWO Baseline steel tube  
Floor Panel must have EI more than ONE Baseline steel tube

see (F.7.3.2, F.7.5.3, F.7.5.4)

- Height/Width of Flat Panel cross-section

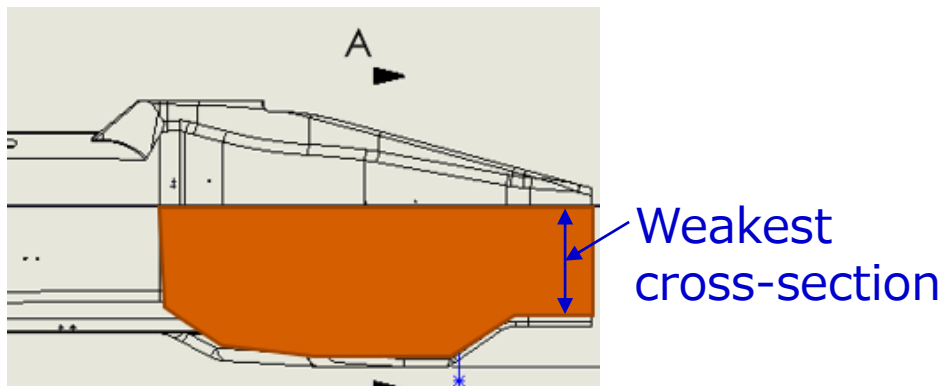
The SES specifies the cross-section as Minimum or average

Select the minimum or average cross section for the FBHS.  
Access holes or single skins are not counted, and usually create the minimum cross section.  
Treat sharp cross sectional discontinuities (example: damper cutouts) like holes.

## FSAEJ strongly recommends to select MINIMUM

Reason

- If they are equivalent at the weakest cross section, the whole is more than equivalent.
- the average requires more complex calculations than select the minimum.
- When you select Average, the Minimum is not equivalent



If Average cross section is selected,

**It must be determined from the integrated area value.**

In the most case... **(Max.+Min.)/2≠Average**

Describe the calculation process in the SES.

## Monocoque SES

### F.4.3 Composite

Attention : Rule References in the SES are not reflected changes in 2025 Rules  
This guidance is based on description of the SES

2025 v1.1 Compliant

# F.4.3 Composite

- Derivation of Key Elements for Proof of Equivalence

If it is a tube frame, it is possible to use common values for physical properties such as Young's modulus and yield strength, but for Monocoque, physical properties vary greatly depending on how it is made, so it is essential to derive physical properties through actual tests.

Reuse of test results from different years is prohibited ( F.4.2.1b ).

## Contents of F.4.3 Composite

- ① Use of Laminates
- ② 3-point Bending of Size-B Steel Tube(s) F.4.2.3, F.4.2.4
- ③ Laminate Test (3-Point Bending) F.4.2.2, F.4.2.4
- ④ Derived physical property value for F.7 (E · UTS)
- ⑤ Shear strength(SIS/FBHS/Acc.Protection/Attachment) F.4.2.5
- ⑥ Shear & Peel strength of adhesion F.4.2.6

Select the Purpose of the Laminate correctly

The image shows a collage of technical documents and test results. Red boxes and numbers 1-6 highlight specific sections:

- 1**: A table with columns for material properties and test results.
- 2**: A diagram of a 3-point bending test on a steel tube, with text: "CRUSHING HOLLOW 3-POINT TEST RIG END SUPPORTS AND APPLICATOR IS THE MOST COMMON SOURCE OF TEST RIG COMPLIANCE."
- 3**: A text box stating: "Required: Test setup images, measurements. All test samples must be presented at Technical Inspection. Image: measuring skin thickness in mm."
- 4**: A table with columns for material properties and test results.
- 5**: A graph titled "Perimeter Shear Test" showing a plot of Displacement vs. Force.
- 6**: A diagram of a shear and T-peel test, with text: "SHEAR AND T-PEEL NEED TO TEST THE SAME BOND OVERLAP USED IN THE CHASSIS."

If you have Different Layup, duplicate this sheet.

# F.4.3 Composite

## ① Use of Laminates

Almost Skin thickness in SES must be described by Scaling option (Integer only)

Layup Used: **SIS F.4.3 Composite** EQ  
**Monocoque** EQ  
 Core thickness: **12** mm EQ  
 Outer skin thickness: 3 mm EQ  
 Inner skin thickness: 3 mm EQ  
 Panel thickness: 18 mm EQ

Scaling option, layup repeats: **1** EQ  
 Scaling option, layup repeats: **1** EQ

Must be an integral multiple of the Layup Schedule for Laminate Test (Because thickness change of 1 ply unit may lose quasi-isotropy)

For example, if you subtract 1ply from [0/45/90/-45] evenly



Reduced stiffness in specific directions

±60deg or 90deg direction need 50% or more at 0deg (See Comment in SES)

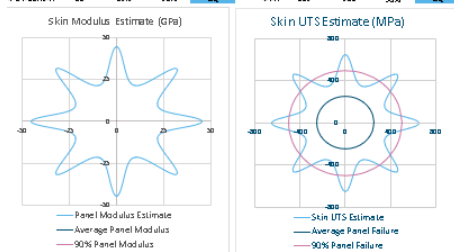
All thickness differences that are not integral multiples should be treated as Different Layups and their physical properties should be obtained using the Laminate Test.

Material	Direction (degrees)	Fibers in ply	Weight (g/m <sup>2</sup> )	Instance
Carbon	0	Biaxial - Perpendicular Balance	200	2
Carbon	45	Biaxial - Perpendicular Balance	200	2
Carbon	90	Biaxial - Perpendicular Balance	200	2
Carbon	-45	Biaxial - Perpendicular Balance	200	2

F.4.2 Two basic quasi-isotropic layups are [0/-45/90/45] and [0/90/-45]. Quasi-isotropic is used to describe laminates that have identical tension modulus in the directions checked using matrices. In practice, the reductions between measured angles is ignored for quasi-isotropic layups.

F.2.1.2.b The SES defines the angles where F.4.3.6.c is enforced. For 2023, orientations in either the 90 or 0/-45/90 directions must be at least 50% of the 0 direction. Estimates below are scaled to 100% in the 0 direction. Extremely low values discouraged. Many sensible and acceptable layups since 2019 returned a CHECK readout (below 50% in some other direction). A skin of [0<sub>u</sub>/45/90<sub>u</sub>/45] fails G.1.A Good Engineering Practice. Unillias on their own have low stability. For thin skins, balanced alios generally enclose unit and increase orientations in non-unidirectional directions.

Direction	SKIN STIFFNESS ESTIMATE (GPa)	SKIN STRENGTH ESTIMATE (MPa)
0	44.9 100%	0 552 100%
45	44.9 100%	45 552 100%
90	44.9 100%	90 552 100%
135	44.9 100%	135 552 100%
22	23.5 52%	113 361 58%

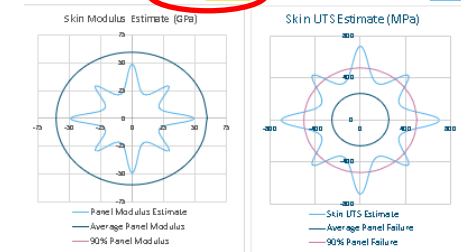


Material	Direction (degrees)	Fibers in ply	Weight (g/m <sup>2</sup> )	Instance
Carbon	0	Biaxial - Perpendicular Balance	200	2
Carbon	45	Biaxial - Perpendicular Balance	200	2
Carbon	90	Biaxial - Perpendicular Balance	200	2
Carbon	-45	Biaxial - Perpendicular Balance	200	1

F.4.2 Two basic quasi-isotropic layups are [0/-45/90/45] and [0/90/-45]. Quasi-isotropic is used to describe laminates that have identical tension modulus in the directions checked using matrices. In practice, the reductions between measured angles is ignored for quasi-isotropic layups.

F.2.1.2.b The SES defines the angles where F.4.3.6.c is enforced. For 2023, orientations in either the 90 or 0/-45/90 directions must be at least 50% of the 0 direction. Estimates below are scaled to 100% in the 0 direction. Extremely low values discouraged. Many sensible and acceptable layups since 2019 returned a CHECK readout (below 50% in some other direction). A skin of [0<sub>u</sub>/45/90<sub>u</sub>/45] fails G.1.A Good Engineering Practice. Unillias on their own have low stability. For thin skins, balanced alios generally enclose unit and increase orientations in non-unidirectional directions.

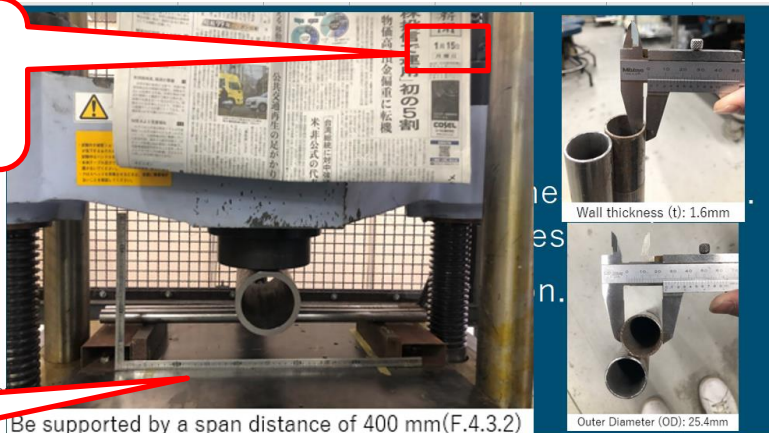
Direction	SKIN STIFFNESS ESTIMATE (GPa)	SKIN STRENGTH ESTIMATE (MPa)
0	48.7 100%	0 702 100%
45	41.1 84%	45 591 98%
90	48.7 100%	90 702 100%
135	41.1 84%	135 591 98%
246	23.5 48%	113 360 54%





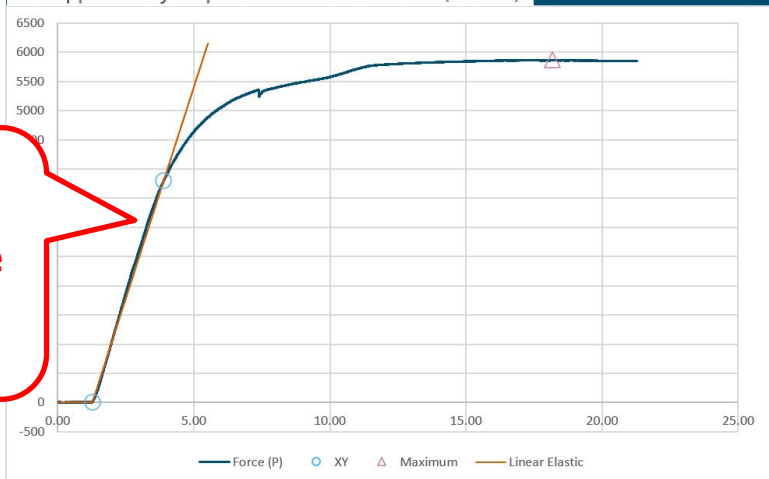
## ② Steel Tube 3-Points Test - How to describe -

Take a photo with the date in the photo (News Paper is the BEST)



Simultaneous Shooting of the Rig and Scale

The Chart is drawn automatically  
Other Chart (your own made or custom) must be REJECT as Format mismatch



EQ Steel Tube 3-Point Test

Paste in logged data from test below:  
Use mm and N, paste values only.  
It is acceptable to resample the data at a lower frequency to reduce the number of datapoints.  
Repeat the energy calculation in column three.  
The graph should automatically generate.  
The formulas should automatically propagate.

MAX	MAX	19	LINEAR
21.282	5865.313	88.50	1.45E+03
mm	N	J	-1.85E+03
Disp. (d)	Force (P)	Energy	Modulus
0.00	0.00	0.00	0.00
0.00	0.00	0.00	0.00
0.00	1.88	0.00	0.00
0.00	0.31	0.00	0.00
0.00	0.00	0.00	0.00
0.00	0.94	0.00	0.00
0.00	1.56	0.00	0.00
0.00	2.19	0.00	0.00
0.00	2.19	0.00	0.00
0.00	2.81	0.00	0.00
0.00	2.50	0.00	0.00
0.00	3.44	0.00	0.00
0.00	1.88	0.00	0.00
0.00	2.19	0.00	0.00
0.00	1.25	0.00	0.00
0.00	2.50	0.00	0.00
0.00	1.88	0.00	0.00
0.00	2.19	0.00	0.00
0.00	3.44	0.00	0.00
0.00	3.44	0.00	0.00
0.00	2.81	0.00	0.00
0.00	2.81	0.00	0.00
0.00	5.00	0.00	0.00
0.00	4.06	0.00	0.00
0.00	2.81	0.00	0.00
0.00	3.13	0.00	0.00
0.00	1.88	0.00	0.00
0.00	2.50	0.00	0.00
0.00	2.81	0.00	0.00
0.00	1.25	0.00	0.00
0.00	1.25	0.00	0.00

Input Test Data directly Displacement(mm) and Force[N]

The load Data must be zero when Actual is no load.

# F.4.3 Composite

## ② Steel Tube 3-Points Test -Detail-

EQ	
If the steel tube is on another tab, type the name of the tab: EQ	
F.4.3.1	Dates of tests: 2023.12.27 EQ
F.4.3.4.a	Metallic load applicator 50mm (2in radius): 50 mm EQ
	Tube Support Span =400mm L: 400 mm EQ
F.4.3.3.a	Number of tubes =2 n: 2 Round EQ
	Wall thickness (t): 1.2 1.6 mm EQ
	Outer Diameter (OD): 25.0 25.4 mm EQ
	Tube cross sectional area (A): 114 120 mm <sup>2</sup> EQ
	Tube second moment of inertia (I): 8509 8509 mm <sup>4</sup> EQ
Enter exact values for minimum and maximum load/deflection in linear-elastic region	
EQ	
x <sub>1</sub>	0.5 mm EQ y <sub>1</sub> 866 N EQ
x <sub>2</sub>	1.5 mm EQ y <sub>2</sub> 3166 N EQ
F.4.3.3.b	Maximum displacement >=19mm: 20 mm EQ
	Test absorbed energy, 19mm deflection (integral P(d)): 0.00 J EQ
y1 to y2	Displacement from bending (P*L <sup>3</sup> /48*E*I): 0.901 mm EQ
	Local crush (P <sup>2</sup> *R <sup>2</sup> *t/(16*pi*Sy <sup>2</sup> *I <sup>2</sup> )): 0.022 mm EQ
	Displacement from shear (0.5*P*0.5*L*shape/A*0.3*E): 0.032 mm EQ
	Displacement from test rig: 0.045 mm EQ
	Rig compliance: 0.020 mm/kN EQ
	Theoretical Tube E: 2.00E+11 Pa EQ
	Tested Tube E (P/2*L <sup>3</sup> /48*2*I*(x2-x1-crush-shear)): 1.90E+11 95.2% EQ
	Maximum Force (<= 19mm displacement): 6441 N EQ
	Maximum Moment (P * L / 4): 6.44E+02 N*m EQ
	Plastic Deformation Stress = (M * OD/2) / (no. tubes * I): 481 MPa EQ
	Rules Baseline UTS: 365 131.69% EQ

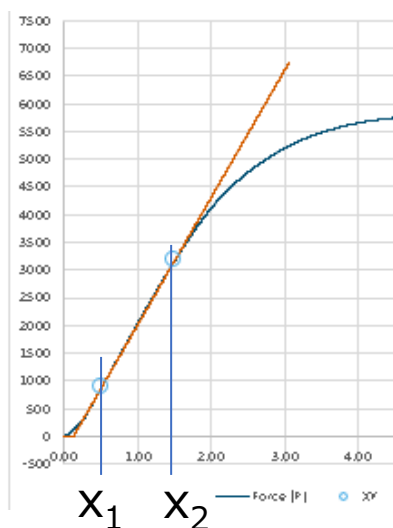
If you have multiple F.4.3 Composite Sheets, you only need to describe it on one sheet.

⇒ For other sheets, just specify that sheet.

x1 : Start displacement of Liner-Elastic Region

x2 : End displacement of Liner-Elastic Region

if you describe it in one sheet  
For Different Layup, just refer to that sheet.



BLANK	
If the steel tube is on another tab, type the name of the tab: EQ	
F.4.3.1	Dates of tests: BLANK EQ
F.4.3.4.a	Metallic load applicator 50mm (2in radius): mm BLANK
	Tube Support Span =400mm L: mm BLANK
F.4.3.3.a	Number of tubes =2 n: BLANK
	Wall thickness (t): 1.2 mm BLANK
	Outer Diameter (OD): 25.0 mm BLANK
	Tube cross sectional area (A): 114 mm <sup>2</sup> BLANK
	Tube second moment of inertia (I): 8509 mm <sup>4</sup> BLANK
Enter exact values for minimum and maximum load/deflection in linear-elastic region	
BLANK	
x <sub>1</sub>	mm BLANK y <sub>1</sub> -2 N EQ
x <sub>2</sub>	mm BLANK y <sub>2</sub> -2 N EQ
F.4.3.3.b	Maximum displacement >=19mm: 20 mm EQ
	Test absorbed energy, 19mm deflection (integral P(d)): 0.00 J EQ
y1 to y2	Displacement from bending (P*L <sup>3</sup> /48*E*I): #DIV/0! mm EQ
	Local crush (P <sup>2</sup> *R <sup>2</sup> *t/(16*pi*Sy <sup>2</sup> *I <sup>2</sup> )): #DIV/0! mm EQ
	Displacement from shear (0.5*P*0.5*L*shape/A*0.3*E): #DIV/0! mm EQ
	Displacement from test rig: #DIV/0! mm EQ
	Rig compliance: #DIV/0! mm/kN EQ
	Theoretical Tube E: 2.00E+11 Pa EQ
	Tested Tube E (P/2*L <sup>3</sup> /48*2*I*(x2-x1-crush-shear)): EQ
	Maximum Force (<= 19mm displacement): 6441 N EQ
	Maximum Moment (P * L / 4): 0.00E+00 N*m EQ
	Plastic Deformation Stress = (M * OD/2) / (no. tubes * I): #VALUE! MPa EQ
	Rules Baseline UTS: 365 #VALUE! #VALUE! EQ

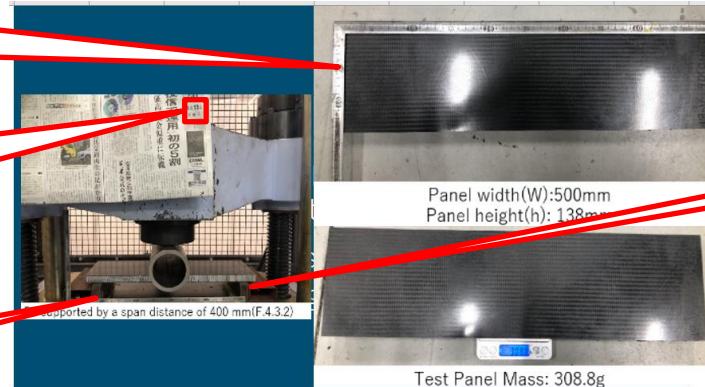


EQ	
If the steel tube is on another tab, type the name of the tab: TestLaminated EQ	
F.4.3.1	Dates of tests: EQ N/A
F.4.3.4.a	Metallic load applicator 50mm (2in radius): mm N/A
	Tube Support Span =400mm L: mm N/A
F.4.3.3.a	Number of tubes =2 n: EQ N/A
	Wall thickness (t): 1.2 mm N/A
	Outer Diameter (OD): 25.0 mm N/A
	Tube cross sectional area (A): 114 120 mm <sup>2</sup> EQ
	Tube second moment of inertia (I): 8509 8509 mm <sup>4</sup> EQ
Enter exact values for minimum and maximum load/deflection in linear-elastic region	
EQ	
0.5	x <sub>1</sub> mm N/A y <sub>1</sub> 866 N EQ
1.5	x <sub>2</sub> mm N/A y <sub>2</sub> 3166 N EQ
F.4.3.3.b	Maximum displacement >=19mm: 20 mm EQ
	Test absorbed energy, 19mm deflection (integral P(d)): 0.00 J EQ
y1 to y2	Displacement from bending (P*L <sup>3</sup> /48*E*I): 0.901 mm EQ
	Local crush (P <sup>2</sup> *R <sup>2</sup> *t/(16*pi*Sy <sup>2</sup> *I <sup>2</sup> )): 0.022 mm EQ
	Displacement from shear (0.5*P*0.5*L*shape/A*0.3*E): 0.032 mm EQ
	Displacement from test rig: 0.045 mm EQ
	Rig compliance: 0.020 mm/kN EQ
	Theoretical Tube E: 2.00E+11 Pa EQ
	Tested Tube E (P/2*L <sup>3</sup> /48*2*I*(x2-x1-crush-shear)): 1.90E+11 95.2% EQ
	Maximum Force (<= 19mm displacement): 6441 N EQ
	Maximum Moment (P * L / 4): 6.44E+02 N*m EQ
	Plastic Deformation Stress = (M * OD/2) / (no. tubes * I): 481 MPa EQ
	Rules Baseline UTS: 365 131.69% EQ

# F.4.3 Composite

## ③ Composite 3-Point Tests - How to describe -

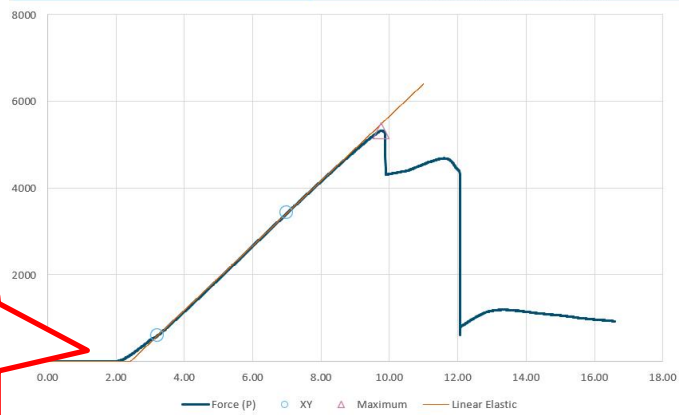
attach a photo showing the dimensions of the Test Piece.



Take a photo with the date in the photo (News Paper is the BEST)

Simultaneous Shooting of the Rig and Scale

The Chart is drawn automatically  
Other Chart (your own made or custom) must be REJECT as Format mismatch



If additional images are necessary, they may be placed below each section.

EQ Composite 3-Point Tests

Paste in logged data from test below:  
Use mm and N, paste values only.  
It is acceptable to resample the data at a lower frequency to reduce the number of datapoints. Repeat the energy calculation in column three. The graph should automatically generate. The formulas should automatically generate.

22.148 mm	9.76 N	22.148 J	LINEAR
Disp. (d)	Force (P)	Energy	Modulus
0.00	0.00	0.00	0.00
0.00	0.00	0.00	0.00
0.00	6.25	0.00	0.00
0.00	4.69	0.00	0.00
0.00	5.00	0.00	0.00
0.00	5.00	0.00	0.00
0.00	5.31	0.00	0.00
0.00	5.63	0.00	0.00
0.00	6.88	0.00	0.00
0.00	5.94	0.00	0.00
0.00	4.69	0.00	0.00
0.00	4.69	0.00	0.00
0.00	4.06	0.00	0.00
0.00	5.00	0.00	0.00
0.00	3.75	0.00	0.00
0.00	3.75	0.00	0.00
0.00	6.25	0.00	0.00
0.00	6.25	0.00	0.00
0.00	5.63	0.00	0.00
0.00	5.31	0.00	0.00
0.00	4.06	0.00	0.00
0.00	4.69	0.00	0.00
0.00	5.31	0.00	0.00
0.00	5.00	0.00	0.00
0.00	5.94	0.00	0.00
0.00	6.56	0.00	0.00
0.00	6.88	0.00	0.00
0.00	6.88	0.00	0.00
0.00	6.25	0.00	0.00
0.00	6.56	0.00	0.00

Same Rig must be set with Steel Tube Test F.4.2.4

Input Test Data directly Displacement(mm) and Force[N]

The load Data must be zero when Actual is no load.

## ③ Composite 3-Point Tests -Detail-

EQ		EQ
F.4.3.1	Dates of tests:	2023.12.27
F.4.3.4.a	Metallic load applicator 50mm (2in radius):	50 mm
F.4.3.2.a	Panel maximum width =500mm (W):	500 mm
	Panel Support Span =400mm (L):	400 mm
	Panel Height = 138mm or 275mm (h):	138 mm
	Centroid from top:	11.500 mm
	Panel Thickness:	23 mm
	Furthest skin centroid (r):	10.750 mm
	Core thickness:	20 mm
	Outer TOP Skin Thickness:	1.5 mm
	Inner BOTTOM Skin Thickness:	1.5 mm
	Second moment of inertia (I):	47921 mm <sup>4</sup>

EQ		EQ
Enter exact values for minimum and maximum load/deflection in linear-elastic region		
x <sub>1</sub>	2 mm	y <sub>1</sub> 2505 N
x <sub>2</sub>	9 mm	y <sub>2</sub> 15763 N
	Force at panel failure:	21045 N
	Absorbed energy at test panel height:	196.23 J
	Single Size B tube gradient:	2415 N/mm
	Minimum panel height for vertical SIS EL, unsealed layup:	175 mm
	Est. panel mass, no resin:	1.04E+02 g
	Test Panel Mass:	200 g
	Est. fiber mass:	103.5 g
	Est Fiber Mass %:	51.75%

A test panel with a gradient  $\geq$  Size B tubing should be stiff enough for equivalence anywhere on the chassis. Derived stiffness is reduced by rig compliance. W76 Tested Tube El < 100% indicates a loss of derived stiffness. The most common source of rig compliance is applicator local crush + end support local crush. Ignore ramp up and fall-off. Select tube and panel x+y from the steepest average sections for best results.

Derived skin modulus of elasticity E ( $\Delta y/L^3/(48*\Delta x)$ ):	52.70 GPa	EQ
Core:	20 mm	EQ
Derived skin modulus of elasticity E:	5.27E+10 Pa	EQ
Outer:	1.5 mm	EQ
Derived skin modulus of elasticity E:	7.64E+06 psi	EQ
Derived UTS of skins ( $L^3*F_r/(4*I)$ ):	4.72E+02 MPa	EQ
Inner:	1.5 mm	EQ
Derived UTS of skins $\sigma_{u15}$ :	4.72E+08 Pa	EQ
Derived UTS of skins $\sigma_{u15}$ :	6.88E+04 psi	EQ

FSAE Rules specify Core Thickness of Test Panel  
It must use the thickest core associated with each skin layup(F.4.2.2d)

Outer side must be upside (Also Actual Test Setup)

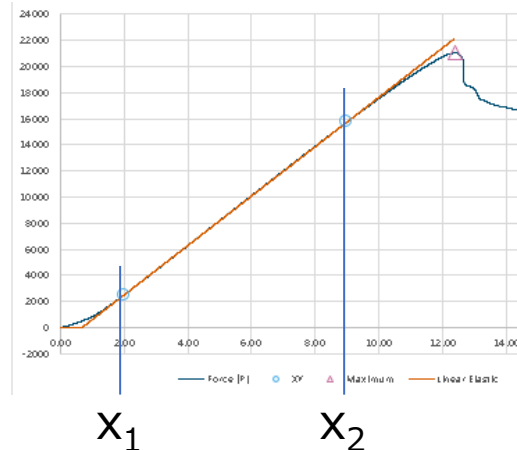
x<sub>1</sub> : Start displacement of Liner-Elastic Region

x<sub>2</sub> : End displacement of Liner-Elastic Region

Measure and record Panel weight before the test

Consistency with the Ply Schedule is checked

④ Young modulus E and UTS are calculated automatically



If Young's modulus/UTS is determined to be inappropriate due to mistake in this sheet, all items that refer to these on other sheets will be **“unconditionally” REJECTED** as unreviewable.

“Unconditionally” means that there is no content review

## ⑤ Perimeter Shear Test

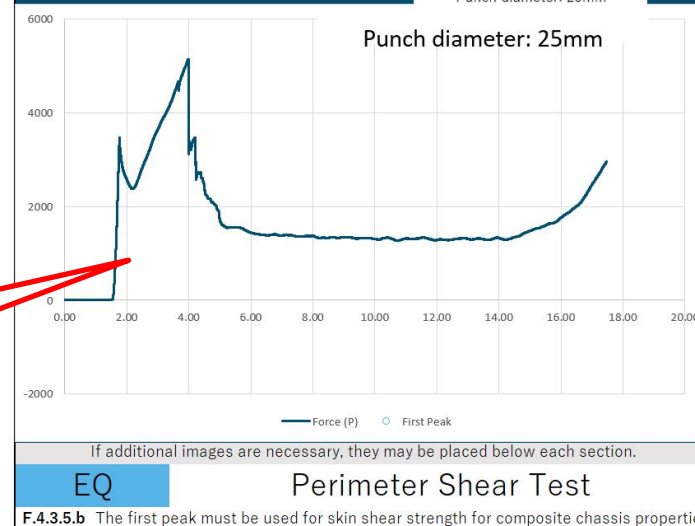
**Test Setup**



Take a photo with the date in the photo (News Paper is the BEST)

Prepare the die with sufficient thickness  
Correct measurement will not be possible if the Bottom Skin touches during the test  
Thin die may be REJECTED

attach a photo showing the dimensions of the Test Piece.



MAX	#N/A
17.46275	5131.658
mm	N
Disp. (d)	Force (P)
0.00	0.00
0.00	0.00
0.00	0.29
0	0.317891
0	0.190735
0	0.095367
0	0.095367
0	0.063578
0	0
0	0.031789
0	0.206629
0	0.333786
0	0.349681
0	0.349681
0	0.429153
0	0.445048
0	0.270208
0	-0.03179
0	-0.22252
0	-0.20663
0	-0.07947
0	0
0.000125	-0.04768
0.000125	-0.14305
0.000125	-0.15895
0.00025	-0.06358
0.00025	-0.01589
0.000375	-0.19073
0.0005	-0.47684
0.000625	-0.68347
0.00075	-0.65168

Input Test Data directly Displacement(mm) and Force[N]

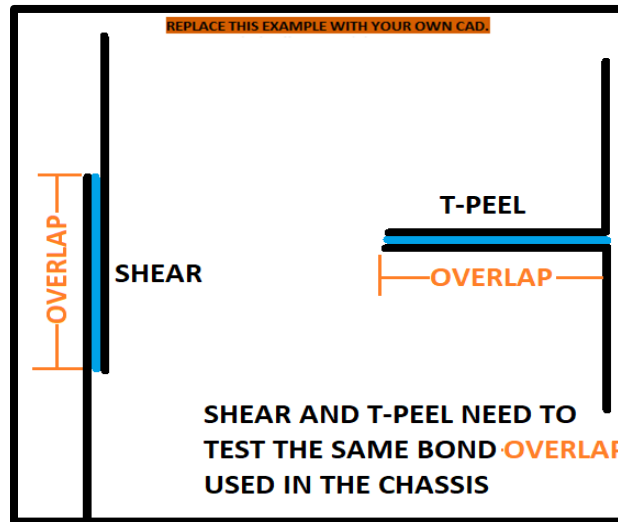
The Chart is drawn automatically  
Other Chart (your own made or custom) must be REJECT as Format mismatch

The load Data must be zero when Actual is no load.



## ⑥ Lap-Joint Test

Enter test results for both Shear Test and T-Peel Test



**Enter pretreatment for adhesion in "Bond prep Process" (e.g. polishing / degreasing)**

### EQ Lap Shear and T-Peel Tests

F.4.3.7.b Samples must be tested in pure shear and T-peel.

EQ			
F.4.3.1	Dates of Shear tests:	<input type="text"/>	N/A
F.4.3.7.a	Shear force at failure or maximum tested force:	<input type="text"/> N	N/A
	Shear test sample lap area:	<input type="text"/> mm <sup>2</sup>	N/A
	Lap Joint Shear Strength:	<input type="text"/> N/mm <sup>2</sup>	N/A
	Lap Joint Shear Strength:	<input type="text"/> Pa	N/A
	Lap Joint Shear Strength:	<input type="text"/> psi	N/A
F.4.3.7.b	Bond overlap length w:	<input type="text"/> mm	N/A
	100% shear strength/unit length:	<input type="text"/> N/mm	N/A
	UTS of skins $\sigma_{UTS}$ :	4.72E+02 N/mm <sup>2</sup>	N/A
	Outer skin thickness:	1.5 mm	N/A
	Load/unit length:	708.154 N/mm	N/A
F.4.3.7.d	Safety Factor	<input type="text"/> mm	N/A

EQ			
F.4.3.1	Dates of T-peel tests:	<input type="text"/>	N/A
F.4.3.7.a	Force at failure or maximum tested force:	<input type="text"/> N	N/A
	T-peel test sample lap area:	<input type="text"/> mm <sup>2</sup>	N/A
	Lap Joint T-peel Strength:	<input type="text"/> N/mm <sup>2</sup>	N/A
	Lap Joint T-peel Strength:	<input type="text"/> Pa	N/A
	Lap Joint T-peel Strength:	<input type="text"/> psi	N/A
F.4.3.7.b	Bond overlap length w:	<input type="text"/> mm	N/A
	100% T-peel strength/unit length:	<input type="text"/> N/mm	N/A
	UTS of skins $\sigma_{UTS}$ :	4.72E+02 N/mm <sup>2</sup>	N/A
	Outer skin thickness:	1.5 mm	N/A
	Load/unit length:	708.154 N/mm	N/A
F.4.3.7.d	Safety Factor	<input type="text"/> mm	N/A

EQ			
F.5.5	0.5 * minimum (T-peel, shear):	<input type="text"/> MPa	
	0.5 * minimum (T-peel, shear):	<input type="text"/> Pa	
	0.5 * minimum (T-peel, shear):	<input type="text"/> psi	

Bond prep process:

## F.4.3 Composite

- Different or Additional Layup

If there are multiple types of layups, duplicate the SES F.4.3 Composite sheet and enter the test results each time

How to refer

### F.7 Composite Chassis

Enter each sheet name in A4:B20 and select from the pull-down menu of each [Layup Used:]

Note: Forces are given in Pa, not Mpa or Gpa.

Material	E (Pa)	S_Ultimate (Pa)
F.3.4.2 Steel	2.00E+11	3.65E+08
TestLaminate1	4.50E+10	4.66E+08
TestLaminate2	5.27E+10	4.72E+08



BLANK Front Hoop Braces (FHB)

The height(d) of the monocoque comparison for Forward FHB must not exceed 50mm.

BLANK

F.6.3 Front Hoop Brace Construction: 0 N/A

Size B Steel Tubes Replaced On One Side: 0 N/A

Layup Used: TestLaminate1 N/A

TestLaminate1 N/A

TestLaminate2 N/A

F.4.3.2.d 50% < Core < 100%: 0.00% Core thickness: EQ N/A

Scaling option, layup repeats: Outer skin thickness: EQ N/A

Scaling option, layup repeats: Inner skin thickness: EQ N/A

Panel thickness: N/A

Half Car Width (Minus holes and single skins): N/A

### F.7.9-10 Attachments / F.8 Front Protection

Enter the sheet name directly in each [Type SES Tab Name Of Layup Used]

EQ

Lap and Anti-Submarine Belt Attachment: 0 N/A

Type SES Tab Name Of Layup Used: TestLaminate1 N/A

Hardpoint type: N/A

---

Monocoque SES

F.7 Composite Chassis

2025 v1.1 Compliant



# F.7 Composite Chassis

- Summary of F.7 Composite Chassis sheet

The image shows a detailed technical drawing of a composite chassis, divided into several sections. The sections are:
 

- Legend Color**: A table defining color coding for different parts.
- Front 3/4 3D CAD** and **Rear 3/4 3D CAD**: 3D models of the front and rear sections.
- Side view** and **Top view**: 2D views of the chassis.
- Dimensions**: A series of tables for FBHS, FHB, SIS, MHBS, ASP, TSP, RIP, and Bottom views.
- EV Protection**: Three tables labeled EV①, EV②, and EV③, detailing accumulator side protection, rear bulkhead support, and rear bulkhead details.

 The drawing includes numerous tables with columns for 'Monocoque' and 'Blank' values. Red circles with numbers 1-6 and EV1-3 are placed over specific areas to indicate where to fill in values. A red box highlights the bottom right section with the text: 'Fill in BLANKs where Selected 'Monocoque' in F.3.1-5 Tube Chassis'.

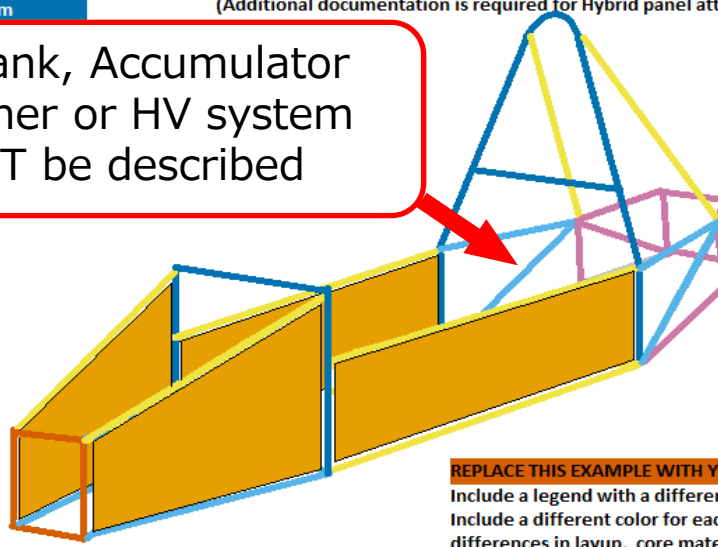
- ① Physical property value for each Layout (Enter F.4.3 Composite sheet name)
- ② Describe the color coding and meaning of colors for each layout of each figure
- ③ FBHS Equivalency (③\* stands for Steering Protection)
- ④ FHB Equivalency (④\* stands for Rearward FHB if necessary)
- ⑤ SIS Equivalency (⑤\* stands for the case of laminated floor)
- ⑥ MHBS Equivalency
- EV①-③ EV Protection structure Equivalency.

- Entry Example of 3/4 CAD

The SES can calculate equivalence for a full monocoque.  
The SES can calculate Hybrid equivalence for panels replacing FBHS and/or SIS diagonals.  
(Additional documentation is required for Hybrid panel attachment.)

25mm x 2.5mm

Fuel Tank, Accumulator Container or HV system MUST be described



Add a side view showing the dimensions of each part as in the example (for smooth inspection)

REPLACE THIS EXAMPLE WITH YOUR OWN

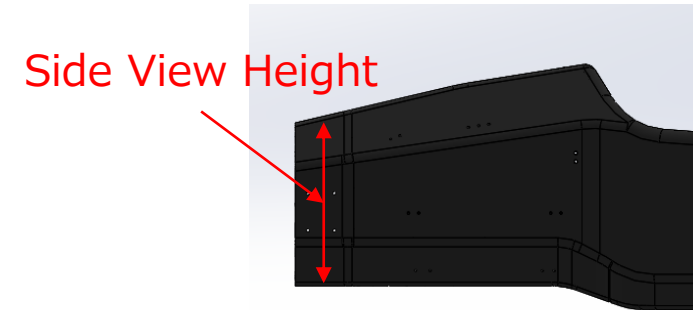
Include a legend with a different color for each tube. Show differences in layup, core material, etc. Show the Fuel / HV system in orange. Use the same color for all tubes smaller than 25mm or 1.2mm (.047in) wall thickness. This is structural (T.2.5.4). Consider making...

1. Show the requested CAD
2. Fuel Tank or Accumulator Container be described
3. It is recommended that the color coding of the pipe is the same as the sample.
4. All pipes with an outer diameter of <25 mm or a wall thickness of <1.2 mm Should be of the same color.

- Front Bulkhead Supports (FBHS)

## (1) Flat panel calculation

- ⇒ Equivalence to 3 Size-B steel tubes is evaluated based on Side View Height
- Indicate that the entered Panel Height should be the weakest dimension
- (If there is an opening, subtract its dimension)



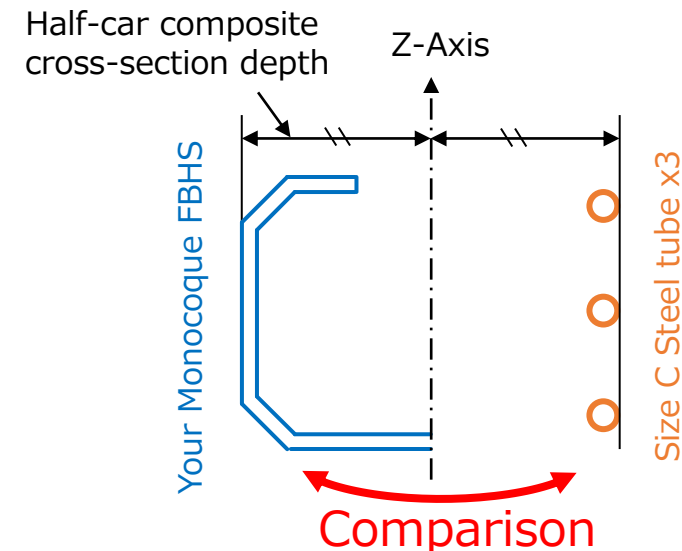
All Dimensions entered to SES must be shown by CAD Images

## (2) If equivalence is less than 100% in (1), use OPTION – Half Car ~

Enter the **Cross-sectional area of skin only** to “Half-car composite cross sectional area”

Enter the **Outer Width from car center axis** to “Half-car composite cross-section depth” (refer to right fig.)

Enter the **area moment of inertia  $I_{zz}$  for only the skin around the vehicle center axis** (Z axis) to “Half-car composite second moment about car centerline”



The actual calculation is to calculate the magnification of the moment of inertia around the Z-Axis when the flat panel is at half-car width, and then multiplying the flat panel by that magnification and comparing it with 3 Tubes. Since it is only about 130% at most, it is desirable to design it so that it can be achieved with a flat panel.

- Supplement of OPTIONAL Calculation

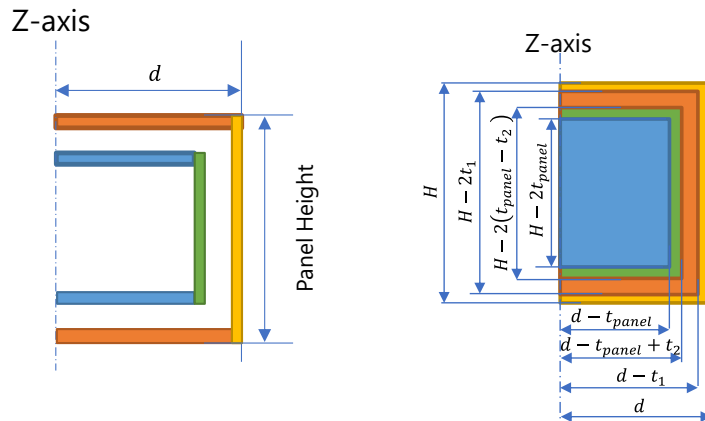
## Rectangle I\_\* estimation

### Maximum rectangular MOI estimation

based on panel specifications, panel height, and Half car width

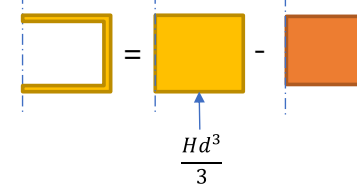
MOI : second Moment of Inertia

Rectangle I\_vertical estimation:  mm<sup>4</sup>

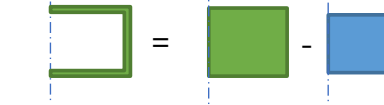


$$I_{ZZ} \text{ estimation} = \frac{Hd^3 - (H - 2t_1)(d - t_1)^3}{3} + \frac{\{H - 2(t_{panel} - t_2)\}(d - t_{panel} + t_2)^3 - (H - 2t_{panel})(d - t_{panel})^3}{3}$$

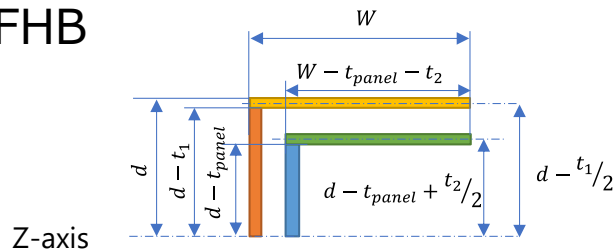
Moment of inertia of outer skin cross-section



Moment of inertia of Inner skin cross-section



For FHB



$$I_{ZZ} \text{ estimation} = \frac{Wd^3 - (W - t_1)(d - t_1)^3}{3} + \frac{(W - t_{panel} + t_2)(d - t_{panel} + t_2)^3 - (W - t_{panel})(d - t_{panel})^3}{3}$$

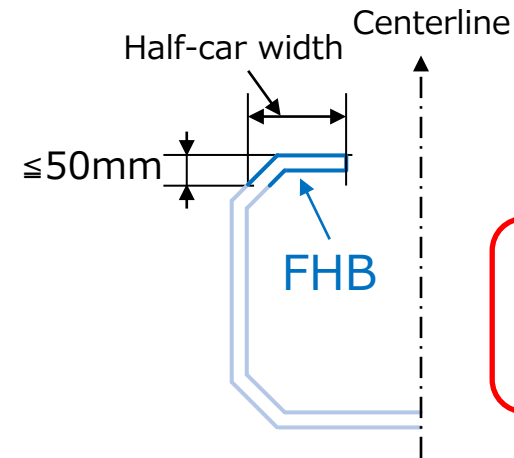
**The actual MOI cannot be greater than the rectangle estimation**

## F.7 Composite Chassis

- Front Hoop Brace (FHB)

### (1) Flat panel calculation

- ⇒ Equivalence to 1 Size-B steel tube is evaluated based on Panel Width
- Indicate that the entered Panel Width should be the weakest dimension
- (If there is an opening, subtract its dimension)



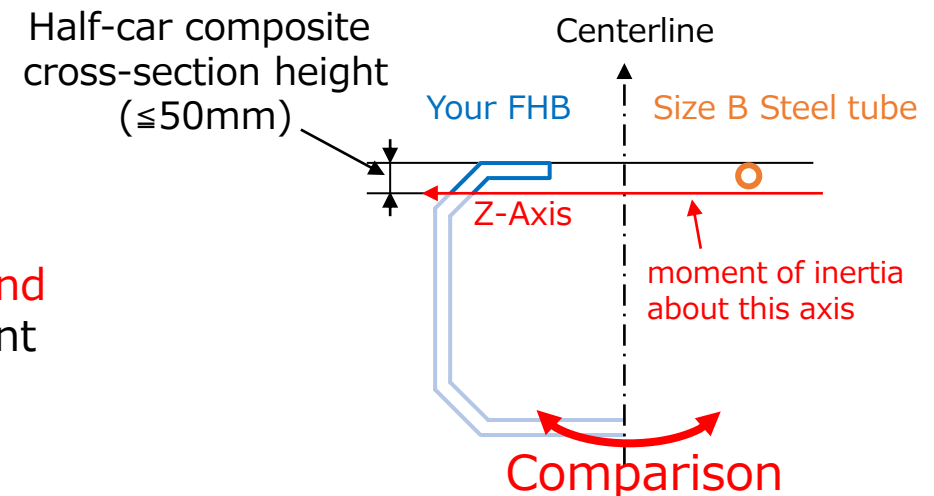
All Dimensions entered to SES must be shown by CAD Images

### (2) If equivalence is less than 100% in (1), use OPTION – Half Car ~

Enter the **Cross-sectional area of skin only** to “Half-car composite cross sectional area”

Enter **FHB height from top (≤ 50mm)** to “Half-car composite cross-section height”

Enter the **area moment of inertia Izz for only the skin around Z axis in right figure** to “Half-car composite second moment about car centerline”



# F.7 Composite Chassis

- Side Impact Structure (SIS)

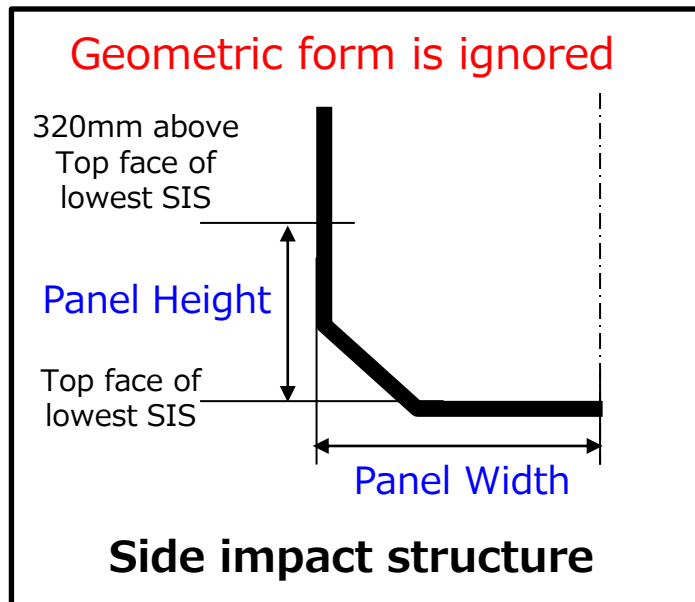
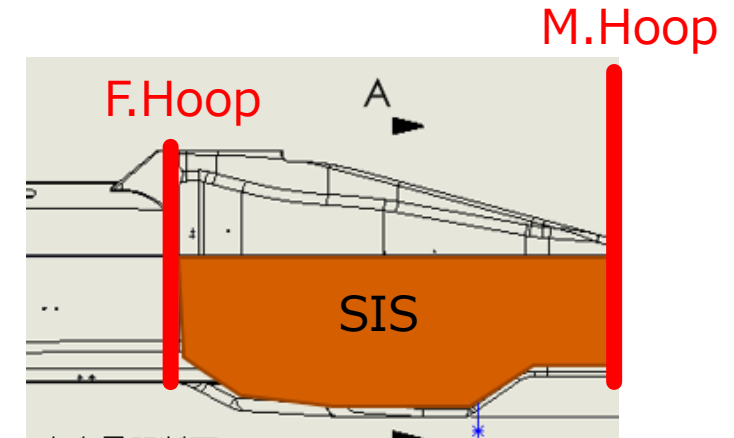
## (1) Flat panel calculation

No allowance for geometric form calculation.(F.4.4)

The following equivalence is evaluated:

- Vertical Wall (Side view height) vs 2 Size-B Steel tubes
- Horizontal Wall (Floor width) vs 1 Size-B Steel tube

If there is an opening, subtract its dimension



Notice : Floor width must be the minimum between FH and MH

FLOOR BETWEEN FH/MH MUST USE MINIMUM, SUBTRACT ALL OPENINGS AND SINGLE SKINS.

**SIS is the most important Driver Protection same as Roll Hoop among Primary Structures, so be sure to prove equivalence based on The Rules!**

All Dimensions entered to SES must be shown by CAD Images

# F.7 Composite Chassis

- Main Hoop Brace Support(MHBS)

## (1) Flat panel calculation

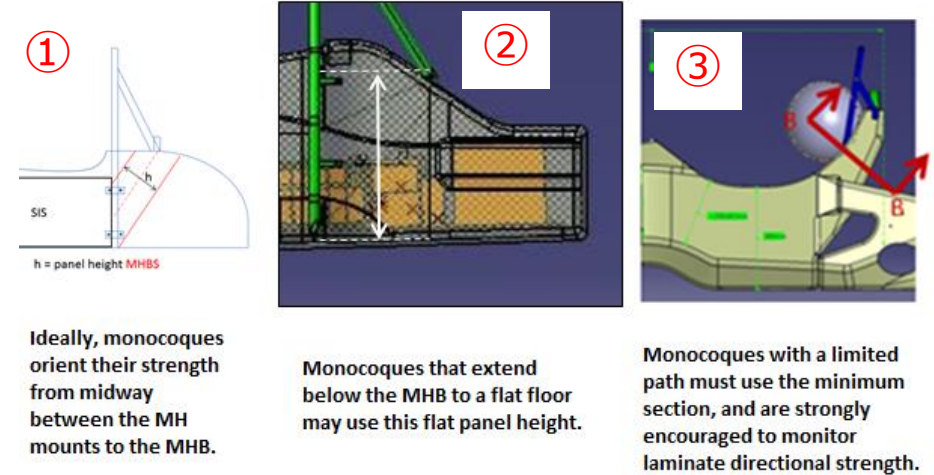
⇒ Equivalence to 1 Size-B steel tube is evaluated based on Panel Width  
 Indicate that the entered Panel Width should be the weakest dimension  
 (If there is an opening, subtract its dimension)

Usually use the **dimension h of ①**

Monocoque that extend below the MHB to flat floor may use Panel Height of ②

③ refer to (2) OPTION calculation

**(2) If equivalence is less than 100% in (1), use OPTION – Half Car ~**  
 Same as FBHS



All Dimensions entered to SES must be shown by CAD Images

- Accumulator Side Protection
- Tractive Side Protection
- Rear Impact Protection

### **(1) Flat panel calculation**

⇒ Equivalence to Specified condition is evaluated based on input dimension  
Indicate that the entered dimensions should be the weakest dimension  
(If there is an opening, subtract its dimension)

### **(2) If equivalence is less than 100% in (1), use OPTION – Half Car ~**

Same as FBHS

For Detachable Rear Impact Protection

Enter the structure description in the field. Refer to the Attachment section for How to describe



Monocoque SES

F.7.8-9 Attachments

2025 v1.1 Compliant

• Summary of F.7.8-9 Attachments sheet

**S.Harness Structure (CAD)**  
BLANK  
Shoulder Harness Structure, Inserts  
Strength of S.Harness bar And Strength of Attach point

**Lap & Anti-sub Structure (CAD)**  
BLANK  
Lap, Anti-sub, 7th Point Inserts  
Strength of Lap and Anti-sub Attach point

**F.Hoop mount (CAD)**  
BLANK  
Front Hoop Mounts  
Strength of F.Hoop Mount

**M.Hoop mount (CAD)**  
BLANK  
Main Hoop Mounts  
Strength of M.Hoop Mount

**Hoop Brace mount (CAD)**  
BLANK  
Hoop Brace Mounts, Steering Protect  
Strength of Hoop Brace Mount

**Acc. Attach (CAD)**  
BLANK  
Accumulator To Mount, Hybrid Panels  
Strength of Accumulator Attachment (Chassis Side)

**Bonded Steering Protection Attachment**  
Strength of Bonded Steering Protection Attachment

**Strength of Hybrid Chassis Attachment**

**Test setup images**

**Anti-sub Load deflection Curve**

**Table Frame - Messages columns N/A.**

Actual Test section Of Harness Attachments

Select Structure and fill in BLANKS

## • Harness Attachments

EQ	Shoulder Harness attachment points:	Monocoque	EQ
T.2.6.2	Shoulder harness mount spacing 175mm to 295mm	200 mm	EQ
	Shoulder attachment test angle should be 90 degrees	90	EQ
F.7.10.2.c	Shoulder harness belt angle (θ = parallel to canal)	N/A	N/A
	Shoulder Harness monocoque canal height:	400 mm	EQ
	Shoulder Harness monocoque canal width:	400 mm	EQ
	Shoulder Harness attachment test canal height:	400 mm	EQ
	Shoulder Harness attachment test canal width:	400 mm	EQ
F.7.10.2.a	Minimum distance, fixture to load 125mm (4.92in)	190 mm	EQ
F.7.10.1.a	Force at failure or maximum tested >= 30kN:	31776.525 N	EQ

Load Direction is specified in the Rules

EQ	Lap and anti-submarine attachment	Monocoque	EQ
F.7.10.1.d	Lap and anti-sub share attachment or insert?	Yes	EQ
F.7.10.1.d	Minimum spacing, lap to anti-sub 125mm (4.92in)	mm	N/A
F.7.10.2.a	Minimum distance, fixture to load 125mm (4.92in)	190 mm	EQ
F.7.10.1.d	Force at failure or maximum tested >= 30kN:	31776.525 N	EQ

If Lap and Anti-sub share Attachment ⇒ Yes

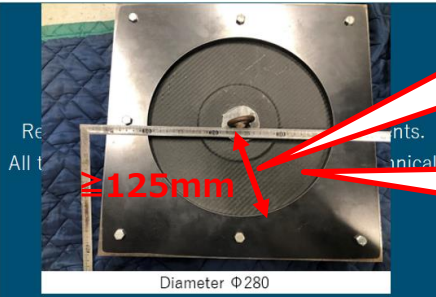
EQ	Separate Anti-Sub	Monocoque	N/A
F.7.10.1.c	Same insert design as lap or anti-sub?	Yes	N/A
	Force at failure or maximum tested >= 15kN:	D N	N/A

If the design of Anti-sub attachment is same as Lap ⇒ Yes

EQ	7th Point Attachment	Six Point Harness	EQ
F.7.10.1.c	Same insert design as lap or anti-sub?		N/A
	Force at failure or maximum tested >= 15kN:	D N	N/A

Tube Frame - Monocoque columns N/A.

The load point must be **no less than 125mm** away from the fixture

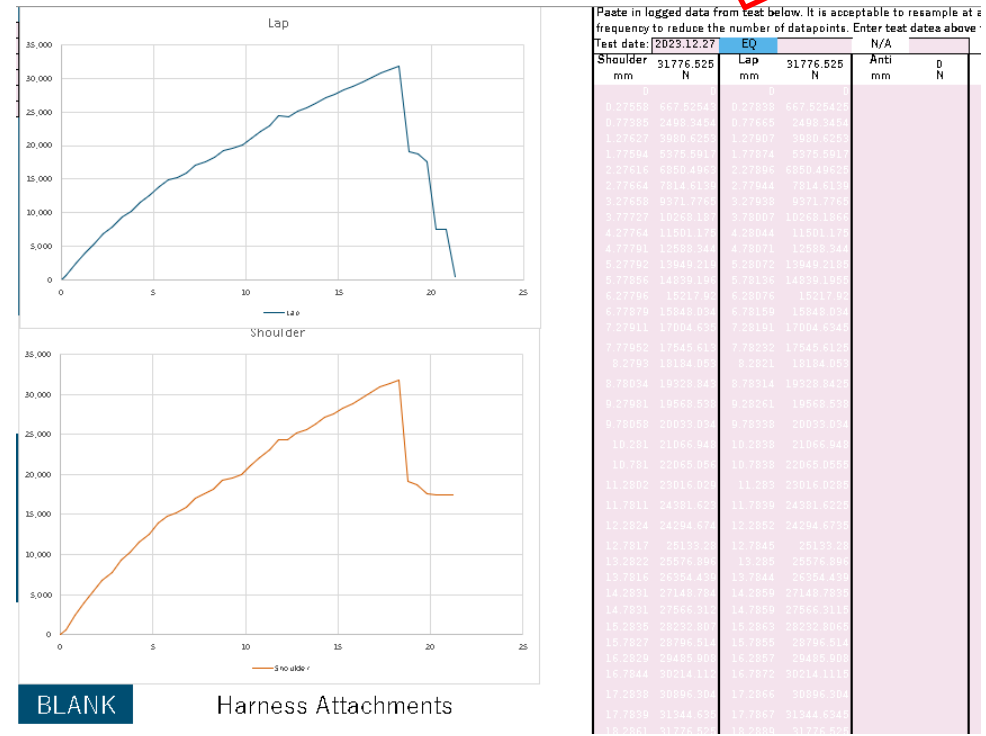


Test panel must NOT be larger than actual size



Test Setup photo should show also test date

Input Test Data directly Displacement(mm) and Force[N]



The load Data must be zero when Actual is no load.

Chart for Shoulder and Lap are automatically created. However, not for Anti-sub, so you must create the Chart yourself if it's necessary

# F.7.8-9 Attachments

- Shoulder Harness Structure, Inserts (Lap, Anti-sub, 7<sup>th</sup> Point Inserts)
  - At Upper part of Shoulder Harness Structure Section, the equivalence to Shoulder Harness Bar (Size-B Steel tube) is evaluated.
  - At Other Section, Pull-out and Tear-out strength of each attachment are evaluated.

## EQ Shoulder Harness Structure, Inserts

For comparison to test results.			
EQ			
F.3.2.1.k	Shoulder Harness Attachment:	Composite	
	Size A Steel Tubes Replaced:	1	
	Type SES Tab Name Of Layup Used:	TestLamina1	
	Core thickness:	20	mm
	Outer skin thickness:	1.5	mm
	Inner skin thickness:	1.5	mm
	Panel thickness:	23	mm
	Composite Panel Dimension (Intersecting Car Centerline):	200	mm
OPTION - Second Moment, Surpassing Flat Panel, CLEAR CELLS IF NOT USED.			
	Car centerline composite cross sectional area (skin only, no core):		mm <sup>2</sup>
	Car centerline composite area moment (I <sub>skinparallel</sub> ):		mm <sup>4</sup>
	1 x Steel Tube	Flat h or L	L <sub>parallel</sub>
F.3.4.1.a	Wall thickness:	0.0024	0.0015
	Outer Diameter / Panel Thickness:	0.025	0.0015
	Cross sectional area (A):	1.73E-04	6.00E-04
	Second moment of inertia (I):	1.13E-08	6.95E-08
F.3.4.2a	Young's Modulus (E):	2.00E+11	5.27E+10
	Ultimate Tensile Strength (S):	3.65E+08	4.72E+08
	Shear:	2.11E+08	9.14E+07
Buckling Modulus	E <sub>1</sub> *I <sub>1</sub> <= E <sub>2</sub> *I <sub>2</sub> :	2.26E+03	3.66E+03
UTS	S <sub>1</sub> *A <sub>1</sub> <= S <sub>2</sub> *A <sub>2</sub> :	6.91E+04	2.89E+05
Bending	4*S <sub>1</sub> *I <sub>1</sub> /r <= 4*S <sub>2</sub> *I <sub>2</sub> /r:	1.30E+03	1.76E+05
Deflection	Bending <sub>1</sub> /(48*E):	1.20E-02	7.41E-03
Energy	F.4.3.2-3 comparison:	5.80E+01	2.84E+02
Flat Panel Properties		Flat Panel Properties	
Outer (b)	0.2 m	A <sub>1</sub>	3.00E-04 m <sup>2</sup>
Outer (h)	0.0015 m	A <sub>2</sub>	3.00E-04 m <sup>2</sup>
Thickness	0.023 m	y <sub>1</sub>	0.00075 m
Inner (b)	0.2 m	y <sub>2</sub>	0.022 m
Inner (h)	0.0015 m	Centroid	0.012 m
		I <sub>1</sub>	5.63E-11 m <sup>4</sup>
		I <sub>2</sub>	5.63E-11 m <sup>4</sup>
		Ic <sub>1</sub>	3.47E-08 m <sup>4</sup>
		Ic <sub>2</sub>	3.47E-08 m <sup>4</sup>
		Ic <sub>32</sub>	6.95E-08 m <sup>4</sup>

Enter the applicable Laminate sheet name

The length of the short side of the panel should be entered

Structure must follow the Rules For Harness Attachments, equivalence should be proven through actual tests, so if it is filled in, it will be EQ

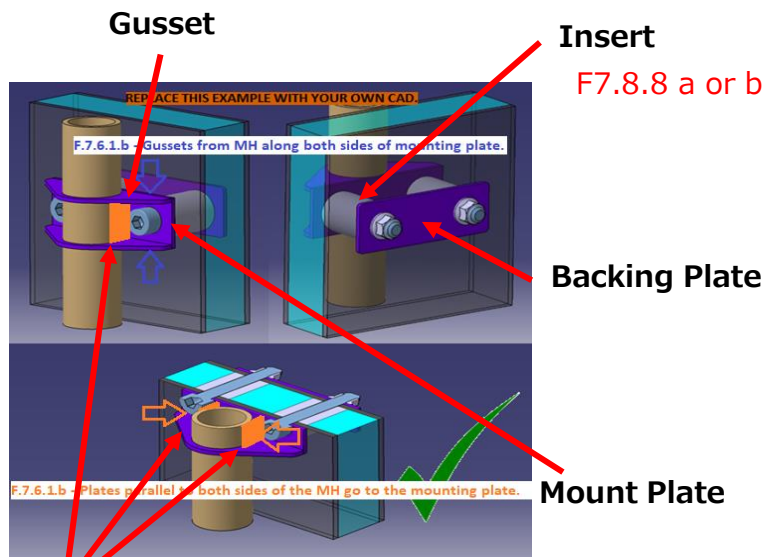
Show each dimension entered to cells

Derived shear strength, for comparison and checking. If single thickness attachments are used, core elimination is expected to be minimized and highly local.

EQ			
	Shoulder Harness Attachment:	Monocoque	
	Type SES Tab Name Of Layup Used:	TestLamina2	
	Hardpoint type:	Skin-Insert-Skin	
	Fastener diameter:	8	mm
	Number of fasteners:	2	
	Panel thickness:	1.5	mm
	Skin thickness - belt side:	0	mm
	Skin thickness - opposite side:	1.5	mm
	Insert material:	Aluminium	Insert thickness:
			0 mm
F.7.8.6	Backing:	Steel	2.00E+11
			1.76E+08 mm
	Backing perimeter on monocoque:	86	mm
	Minimum - Fastener spacing, edge, or corner distance:	90	mm
	Skin shear strength (If tested):	9.14E+07	Pa
	Harness test load / shear area = Min shear strength:	2.46E+08	Pa

# F.7.8-9 Attachments

- Attachment Calculation



Parallel Plate (Steel t2 or thicker)  
M.Hoop & F.Hoop Attachment  
MUST satisfy F.7.6.1

All Dimensions entered to SES must be shown by CAD Images

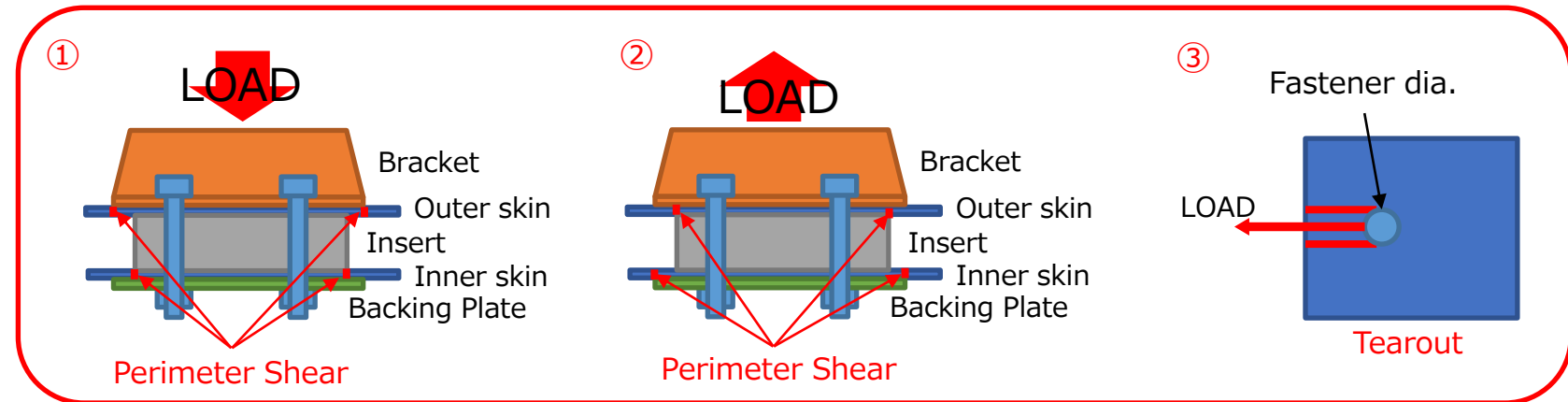
Enter the applicable Laminate sheet name

EQ	F.7.9 Type SES Tab Name: Bottom FH Attachment Layup:	TestLaminate2	EQ
F.7.8.8	Bottom MH Attachment:	Skin-Insert-Skin	EQ
	Fastener diameter:	8 mm	EQ
F.7.8.5 or 7	No. of fasteners (2 x 8mm):	2	EQ
	Panel thickness:	3 mm	EQ
	Insert material:	Aluminium	EQ
	Insert thickness:	0 mm	EQ
	Scaling option. layup repeats:	1	EQ
	Outer skin thickness:	1.5 mm	EQ
	Scaling option. layup repeats:	1	EQ
	Inner skin thickness:	1.5 mm	EQ
	Tube gap from panel:	5 mm	EQ
F.7.7.3	Bracket thickness:	2 mm	EQ
	Steel perimeter on outer skin:	180 mm	EQ
	Insert Perimeter on monocoque:	200 mm	EQ
F.7.8.6 Backing:	Steel 2.00E+11 1.76E+08 mm:	2 100.00%	EQ
	Steel perimeter on inner skin:	180 mm	EQ
Min - Fastener spacing, edge, weaker layup, or corner distance:		60	EQ
	Skin shear strength:	9.14E+07 Pa	EQ
F.7.9.1	① Perimeter shear strength >45000N:	5.21E+04 115.72%	EQ
	② Perimeter shear strength >45000N:	5.21E+04 115.72%	EQ
	③ Tearout strength >45000N:	6.58E+04 146.17%	EQ

Laminate Type(F.7.8.8a or b)

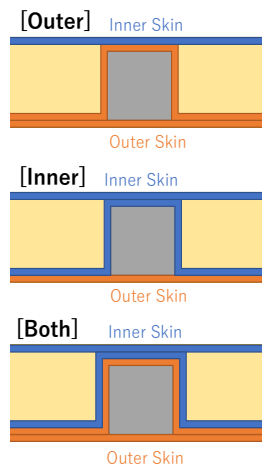
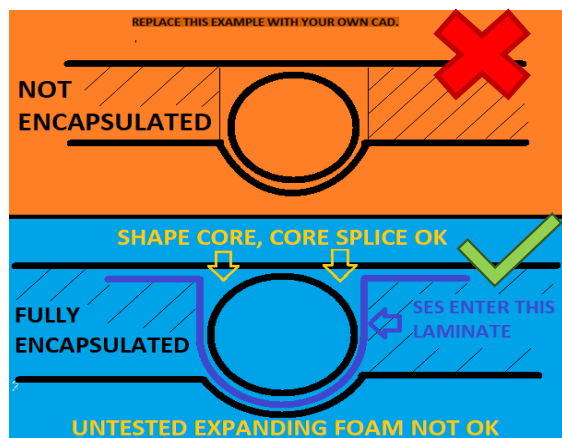
Structural Dimensions

Distance to the nearest Panel Edge



# F.7.8-9 Attachments

- Full-Laminated Front Hoop



Photograph FH welds before laminating. Resubmit SES with FH lamination photos after manufacturing.

F.7.4.3.c Leave 25mm at one end of the Front Hoop exposed to allow diameter inspection.

F.5.7.7 Aluminum FH: Drill a 4mm inspection hole through one side for thickness inspection.

Fully encapsulate the Front Hoop from outer skin to outer skin, and from inner skin to inner skin.

2025 The laminate used for equivalence runs from one skin, around the FH, and back.

BLANK			
F.7.5	Front Hoop Mounts:	Composite	EQ
	Front Hoop Mounts:	Laminated	EQ
	Type SES Tab Name Of Layup Used:	Test_Laminate1	BLANK
	Front Hoop centerline length:	mm	BLANK
	Laminate thickness:	2 mm	EQ
	Skin shear area - centerline x 1 thickness:	0 m <sup>2</sup>	EQ
	Skin shear strength:	0.00E+00 Pa	BLANK
F.7.5.2b	Single tearout path >=180000N:	0.00%	EQ
	Front Hoop Lamination:		BLANK
	Lap joint strength:	5.00E+06 Pa	EQ
	Total bond width including both sides of the Front Hoop:	mm	BLANK
	Bond shear area:	0 m <sup>2</sup>	EQ
F.7.5.2b	Bond failure >=180000N:	0 0.00%	EQ

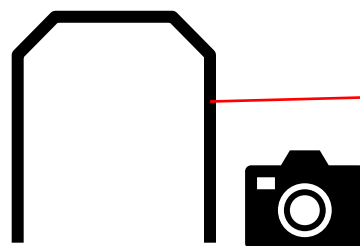
**Not Encapsulated laminated Front Hoop is NOT considered "Full-Laminated"**

Attach a CAD image of the cross-section as the evidence

**Two Photographic Evidences are required**

**They must be attached to resubmit before the submission of shakedown certification**

Before Laminated



Lamination Photo  
Photos of the production process are better

- Insert Material

"MPa" Typo

Insert compressive strength  $\geq 12$  GPa:

$E \geq 4$  GPa:  Shear  $\geq 2.5$  GPa:

Assessing the Insert Material property

They will be ignored 2025 FSAEJ, so enter larger number. However, describe Insert Material in additional info.

### [Guide To 2025 FSAE Frame Rule Changes — DesignJudges.com](https://www.designjudges.com/guide-to-2025-fsae-frame-rule-changes)

Per F.7.8.8, bolted attachment points with test loads (roll hoops, accumulator, AIP, removable rear impact, etc.) require either solid inserts or single thickness layup hardpoints with no core. Inserts provide extra local stiffness as a load path to prevent core crush and engage both skins in the pullout load. You can see that if you work through the SES math. Core material alone does not serve this function. Core alone could allow one skin to be loaded and failed in shear before loading the other skin, basically cutting the load rating of the attachment in half. We have done an initial review of materials. We expect all monocoque teams this year to work toward using inserts that are at least 4GPa modulus, 2.5MPa shear, and 12MPa compression strength. These minimums are effectively End Grain Balsa, but not the engineered balsas whose properties are half as much. We will work with teams and assess the information that comes in this year to write a proper rule for 2026.



End Grain Balsa



Common Balsa  
(Engineered Balsa)

Monocoque SES

F.8 Front Protection

2025 v1.1 Compliant



- Focus on Monocoque (other items are referenced the guidance for F.8)

**FBH CAD dimensions**

**Attenuator and Diagonal**

**Anti-Intrusion Plate**

**IA Attachment, Wing Detachment**

**Front Bulkhead, Composite Diagonal**

**AIP and Diagonal Attachment**

**Physical Test Fixture Guidance**

**Physical Tests**

**Impact Attenuator And / Or Wing Failure Test**

**Composite AIP 120KN Physical Test**

The requirement of Diagonal Tube depends on IA-Type and FBH size.

If Diagonal is required, choose a conformance certification F.8.4.3 method

Front view, shows the front wing and mounts are entirely outside / below the front bulkhead.

Show total number of fasteners for standard shear calculation.

Show fastener UTS source / conversion.

Equivalency of FBH

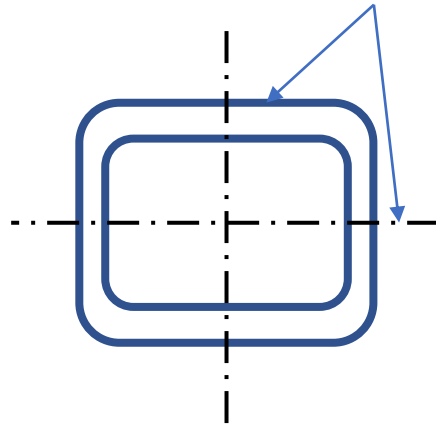
Select Structure and fill in BLANKS

- Front Bulkhead, Composite Diagonal

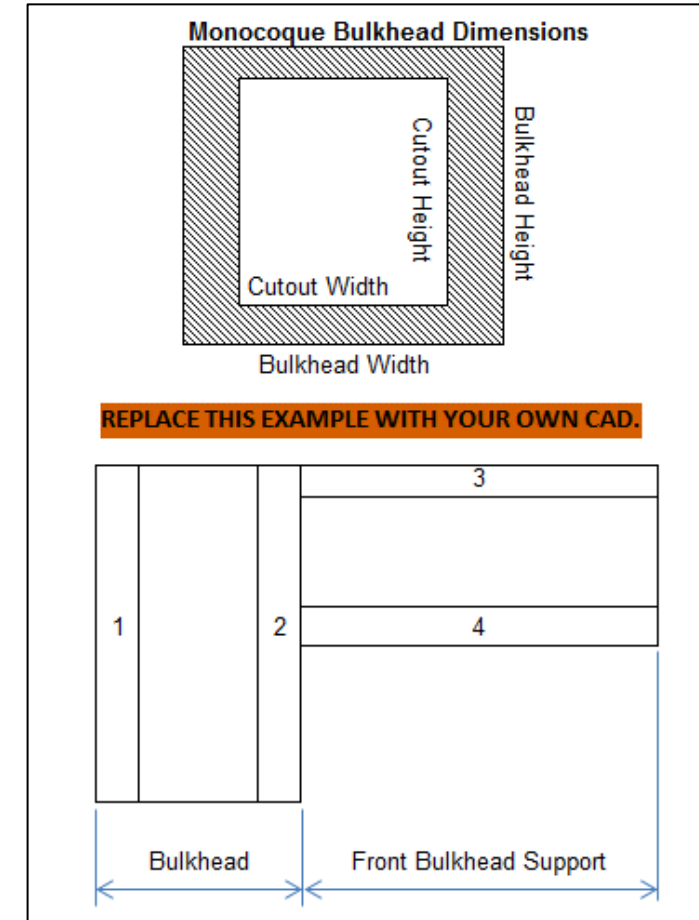
## (1) Flat Panel Calculation

⇒ Equivalence to 2 Size-B steel Tubes is evaluated based on L shape Model

Weaker cross-section is used for calculation



Since the input value of F.7 Composite Chassis is used for the FBHS part, enter them first



F.7.2.1 L shaped Model

